

# Design and certification of the chopper disks for the NEAT II TOF spectrometer: A lesson learned

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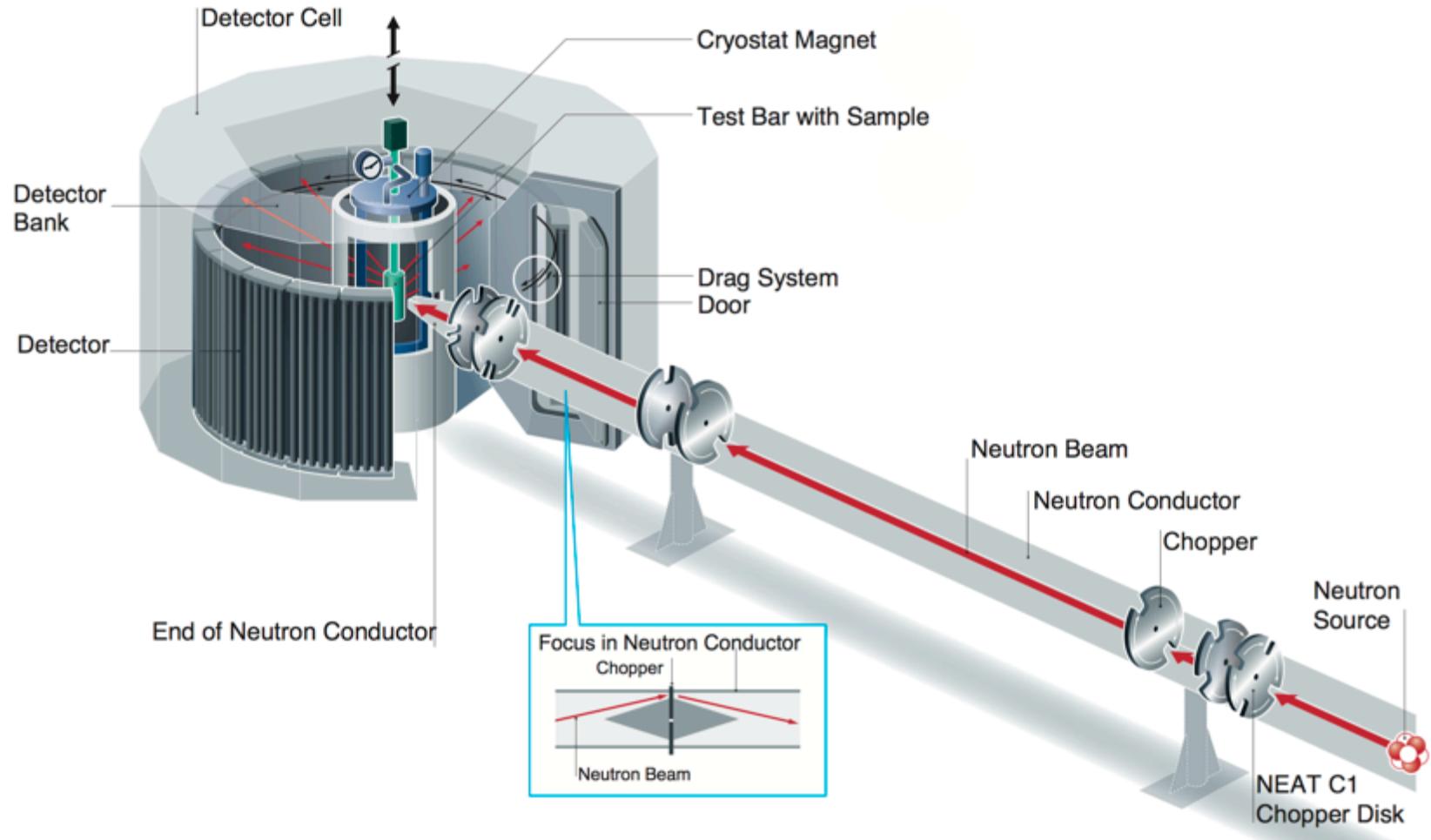
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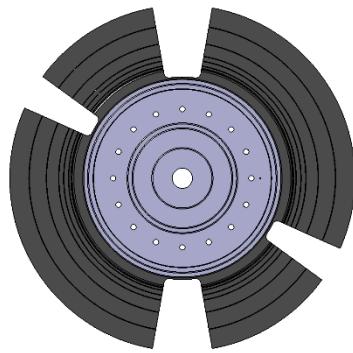
- NEAT II Time of flight spectrometer
- Design loop
- Spin test of the first prototype
- New hub design
- Overspeed test
- Conclusions

# NEAT II Time of flight spectrometer

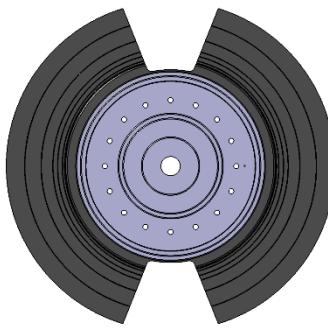


Source: Helmholtz-Zentrum Berlin

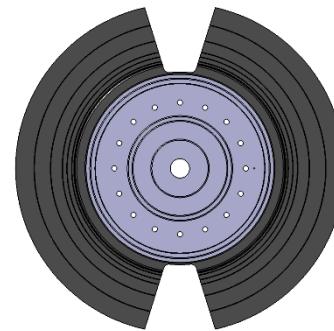
## NEAT II chopper disk series: C1 – C7



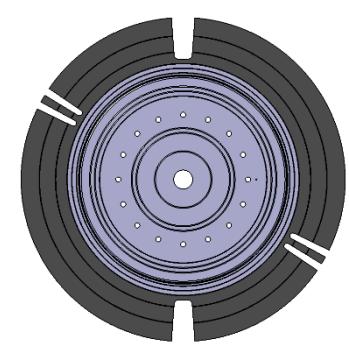
C1 and C2



C3



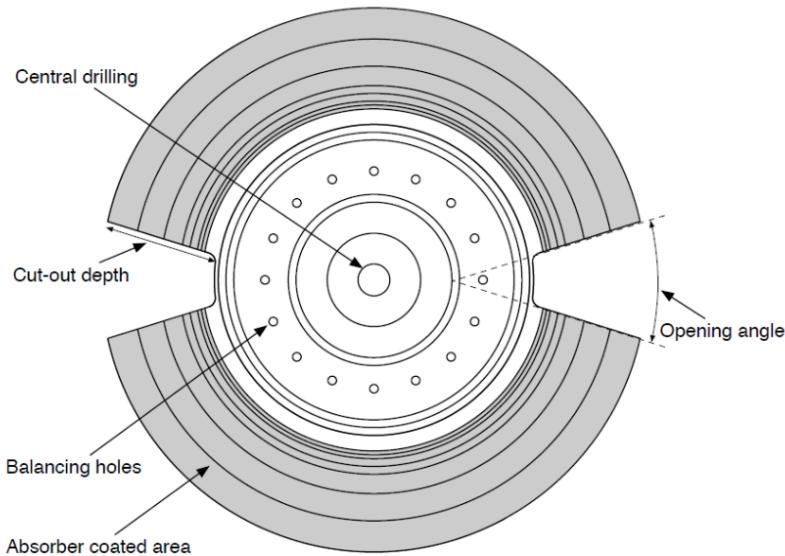
C4 and C5



C6 and C7

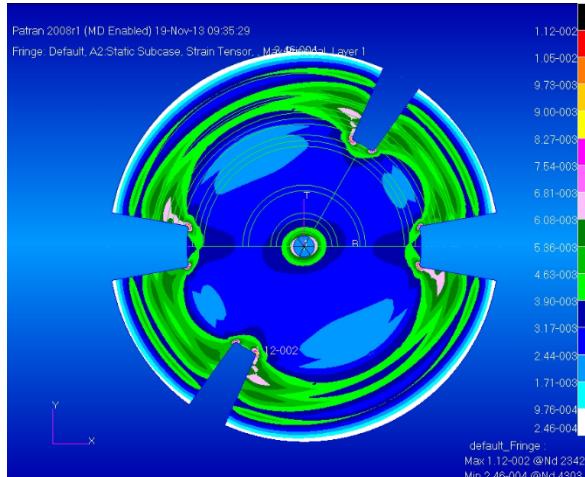
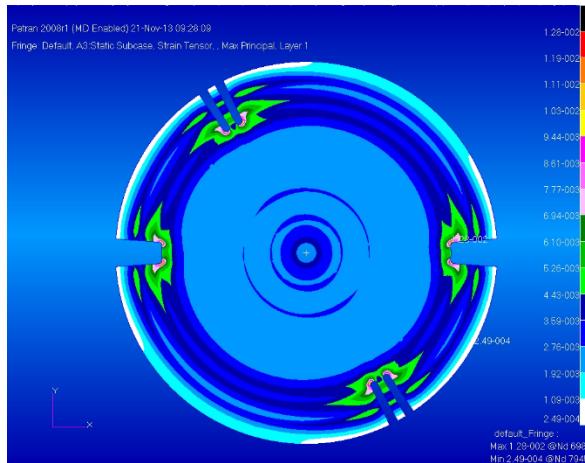
One cross section for all seven disks requires only one mold

# Features of the NEAT II chopper disks

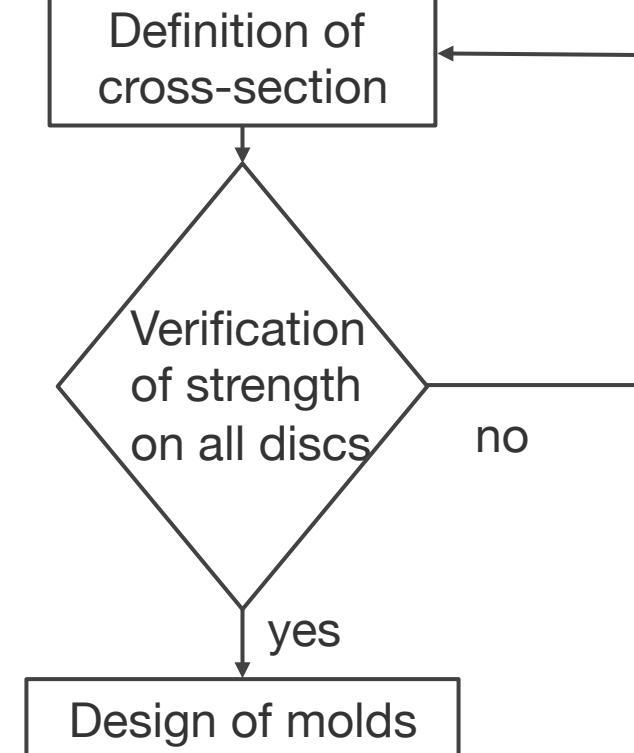


- Diameter: 700 mm
- Cut-out depth: 135 mm
- Operational speed: 20,000 rpm
- Relatively large area with boron coating
- Large opening angle of up to 44°
- Small central drilling
- Unsymmetrical cut-outs (C1/C2)

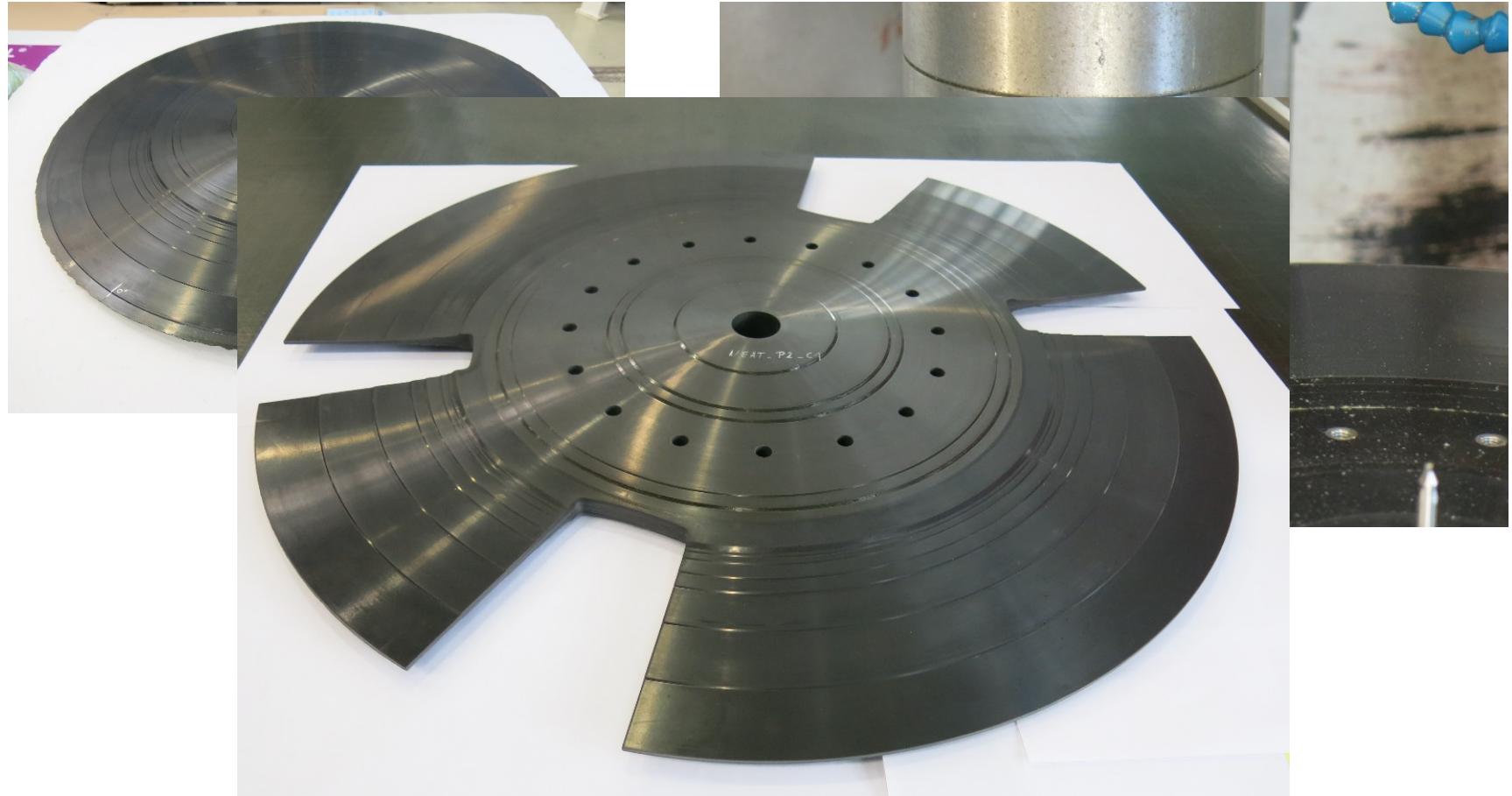
# Design loop



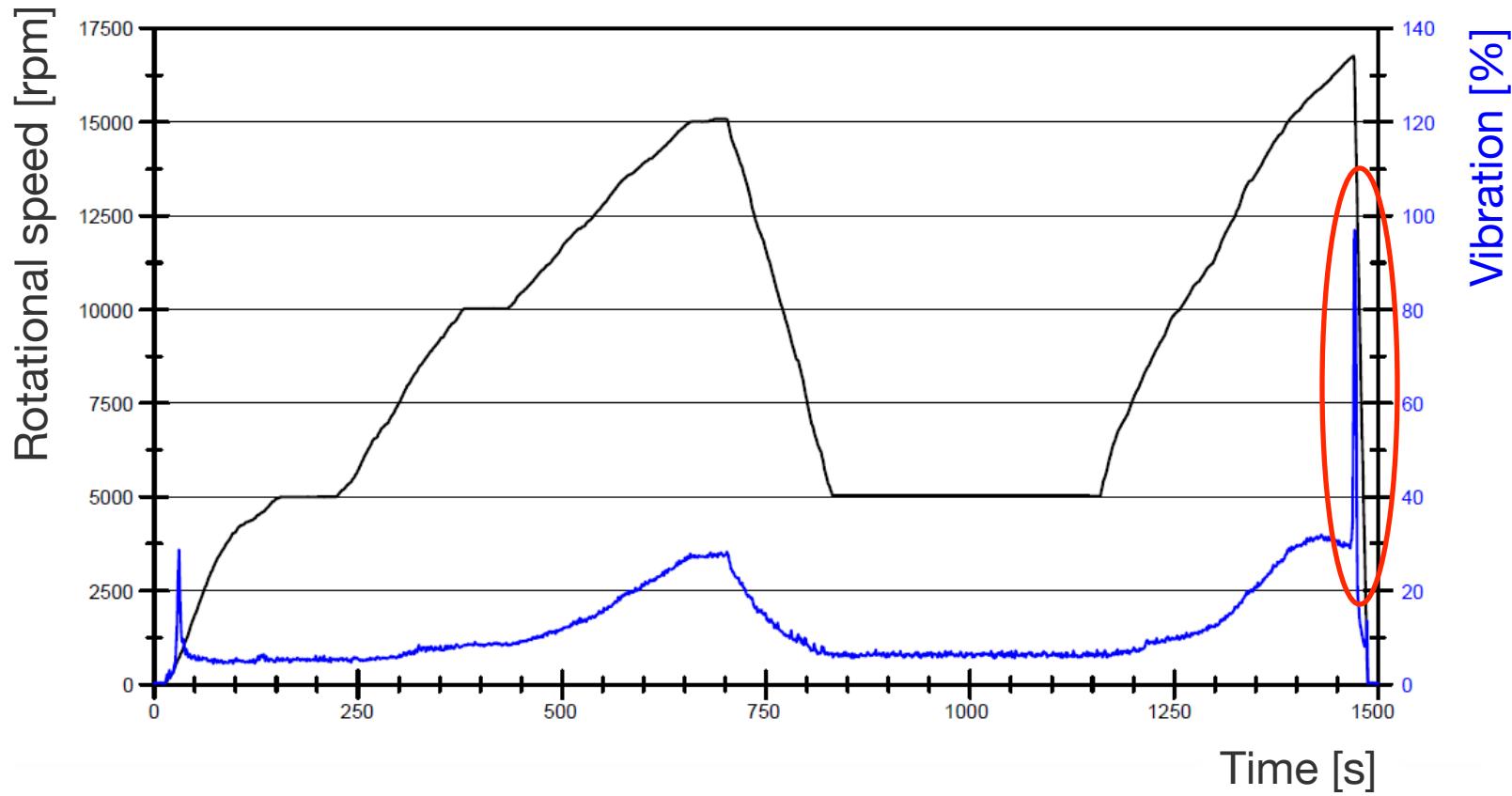
Cross-section optimization of most critical disk (weight/strength)



## P1 – First prototype (C1 design)



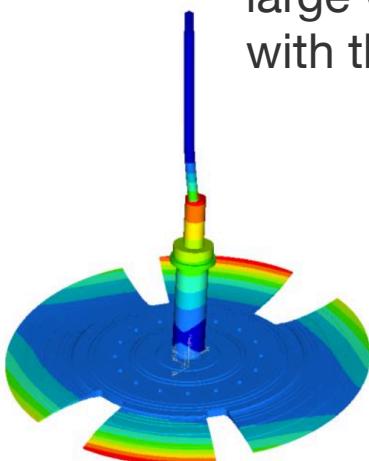
# Spin test of P1



# Possible causes of vibration peak

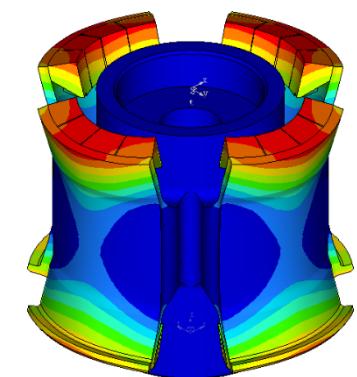
## Disk

- Natural frequencies of the disk get triggered by rotational speed
- Coupling effects of the large wings of the disk with the shaft



## Hub

- Low bending stiffness
- Detachment from disk
- Low contact pressure



# Optimization of thickness distribution

- Blue: Design space
- Gray: Non-design space  
(boron coating)

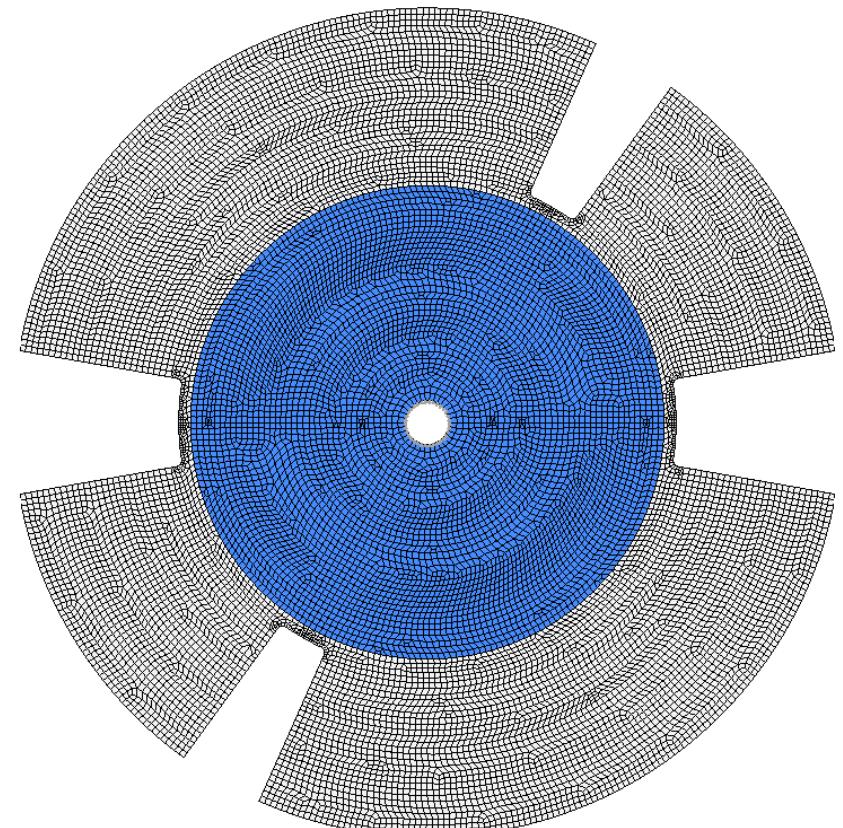
$$\min\{z(H) \mid g_1(H) \leq 0 \mid g_2(H) \leq 0\}$$

Objective function:  $z = -\sum_{i=1}^3 f_{e_i}(H)$  Natural frequency

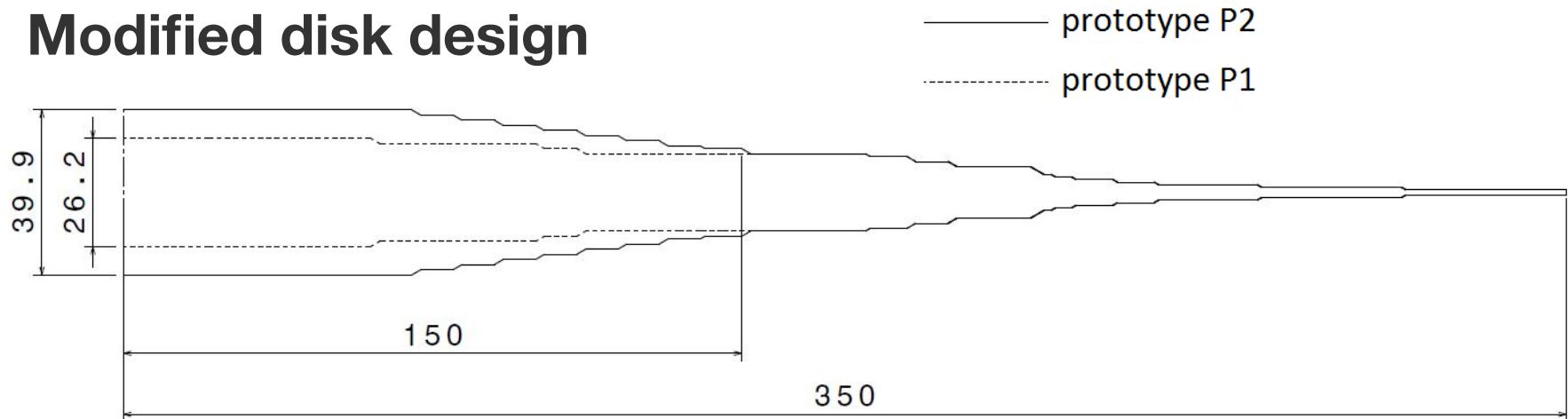
Constraints:  $g_1 = \frac{m(H)}{m_{max}} - 1 \leq 0$  Mass

$$g_2 = \frac{FI}{0.8} - 1 \leq 0 \quad \text{Failure index}$$

Design domain:  $H \in [1.28; H_{max}]$  Element thickness



# Modified disk design



Additional CFRP layers in the central area in order to:

- Stiffen the disk
- Reduce the widening of the central drilling

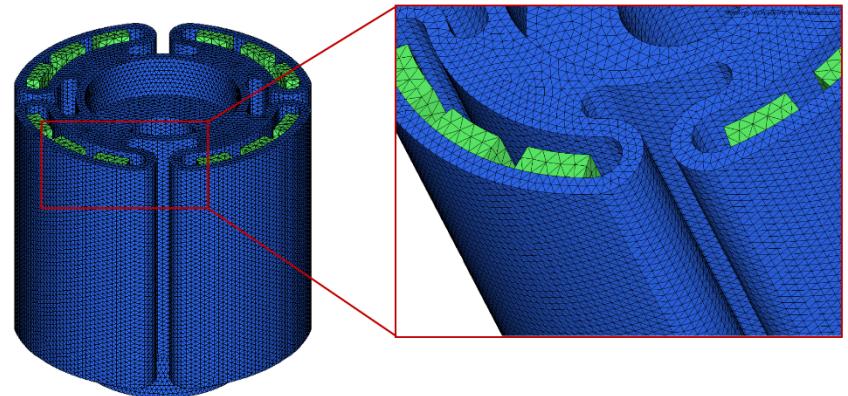
But:

- 20 % weight increase

# New design of a hub with single curved wings



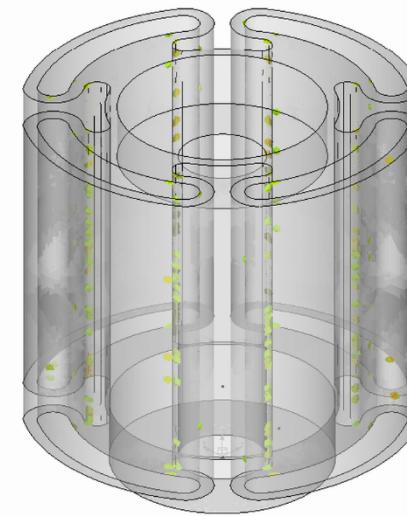
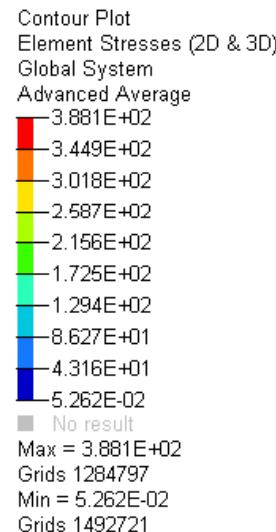
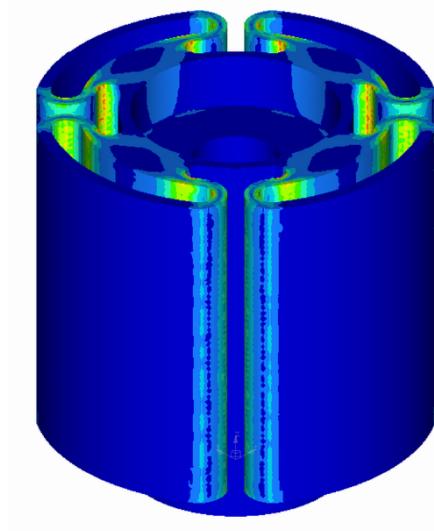
FE model:



|             |                        |
|-------------|------------------------|
| DOF:        | 1.7 m.                 |
| Elements:   | CTERTA, Solid 2. order |
| Constraint: | Inertia relief         |

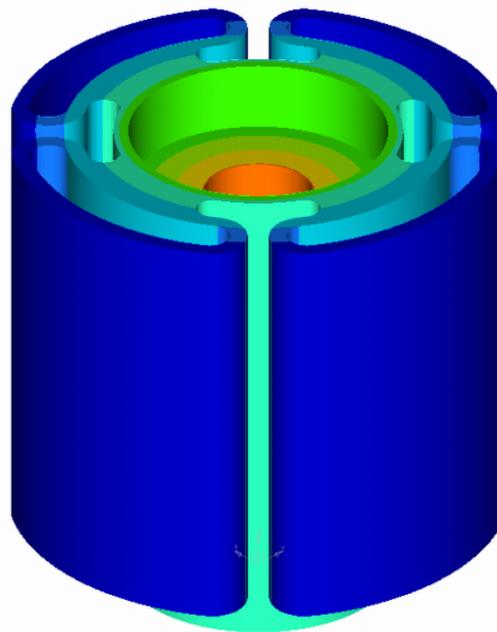
# Avoiding plastic deformation after cold pressing-in

- Stress peaks up to 390 MPa caused by numerical elements (CETETRA)
- Elements with stress > 250 MPa



# Oversize of the hub

Contour Plot  
Displacement(X)  
Analysis system  
0.000E+00  
-1.146E-02  
-2.292E-02  
-3.438E-02  
-4.585E-02  
-5.731E-02  
-6.877E-02  
-8.023E-02  
-9.169E-02  
-1.032E-01  
No result  
Max = 0.000E+00  
Grids 2186089  
Min = -1.032E-01  
Grids 3106992

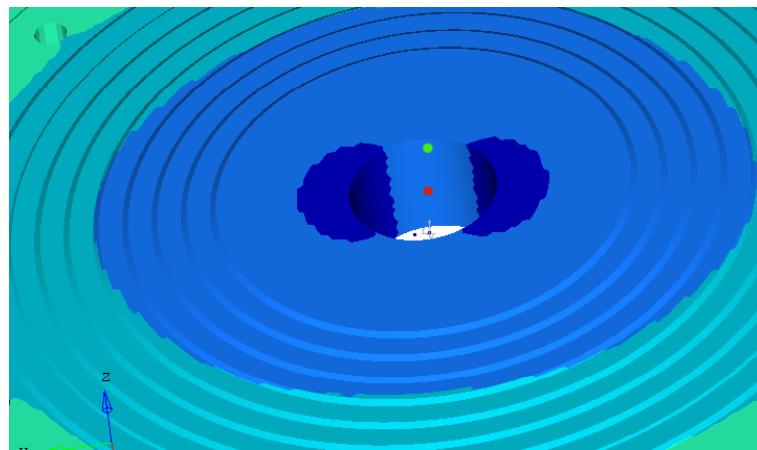


**Radial contraction ( $\Delta T = -180 \text{ K}$ ):**  
 $\Delta r = -0.1 \text{ mm}$

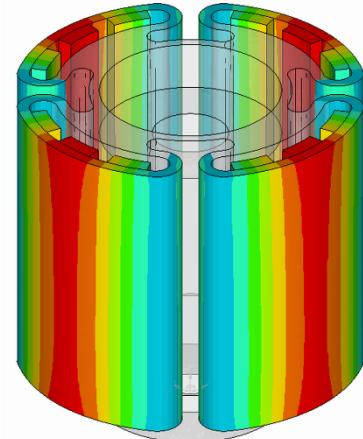
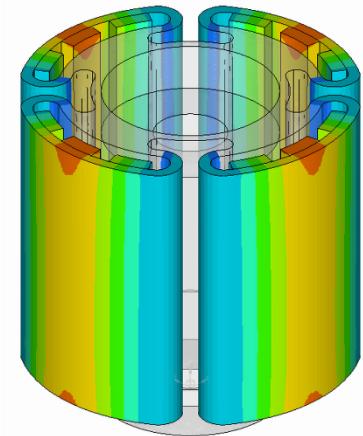
$\varnothing$  Drilling disk P2: 50,07 mm  
 $\varnothing$  Hub at 20 °C: 50,27 mm  
 $\varnothing$  Hub at -160 °C: 50,07 mm

# Radial widening during rotation

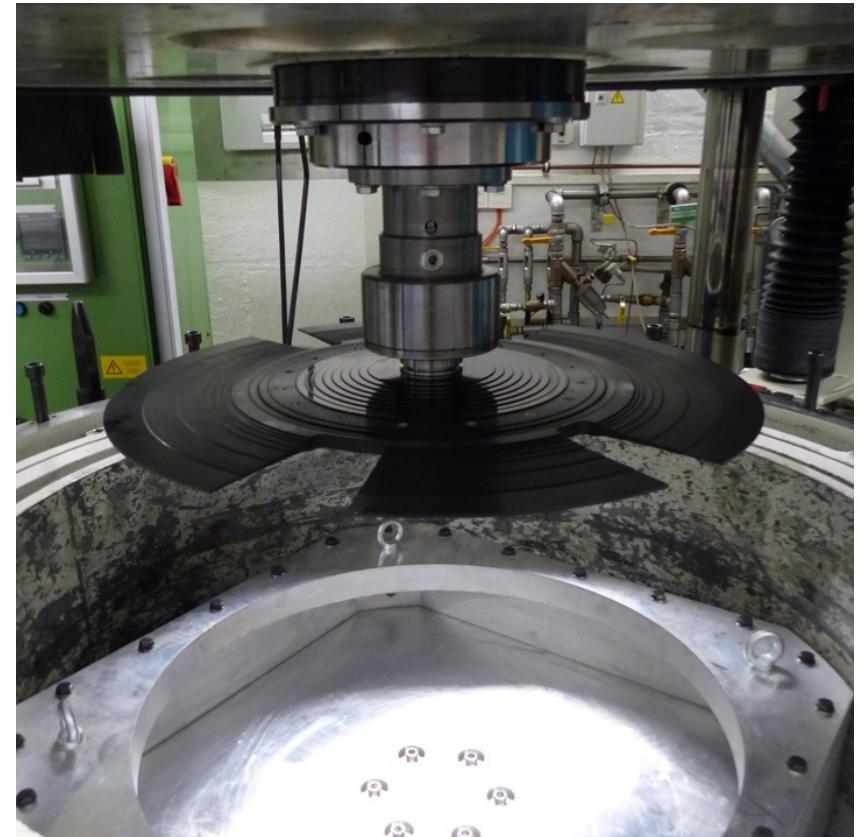
- Oval widening of the central drilling
- Calculating contact area for operational speed and design speed with respect to the prestrain



Contour Plot  
Displacement(X)  
Analysis system  
3.000E-01  
2.667E-01  
2.333E-01  
2.000E-01  
1.667E-01  
1.333E-01  
1.000E-01  
6.667E-02  
3.333E-02  
0.000E+00  
■ No result  
Max = 2.419E-01  
Grids 2192525  
Min = -5.353E-04  
Grids 2208576  
A = 350 Hz (21,000 rpm)

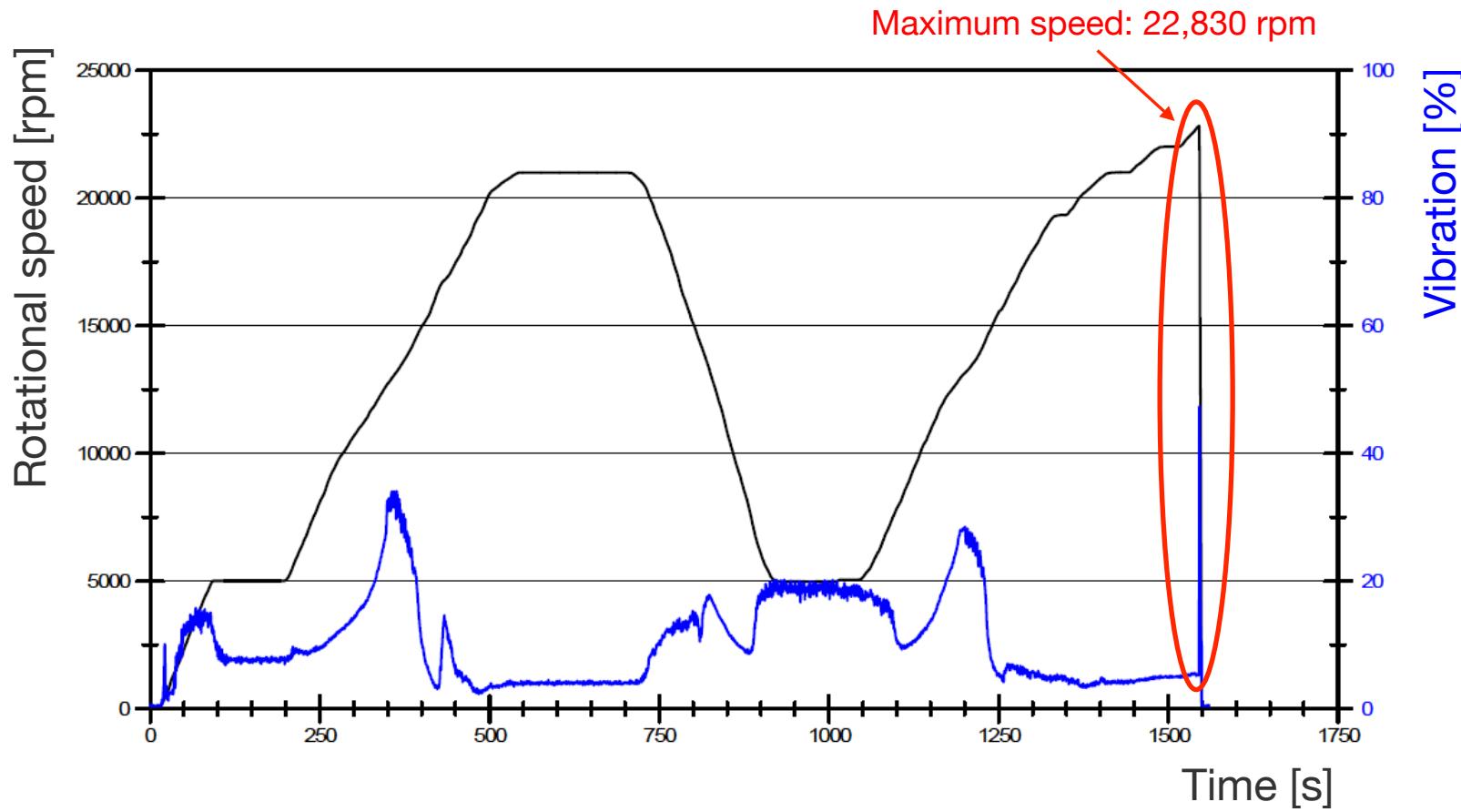


# Overspeed test prototype P2 with new hub design



|                      |                     |
|----------------------|---------------------|
| Operational speed:   | 333 Hz (20,000 rpm) |
| Certification speed: | 350 Hz (21,000 rpm) |
| Design speed:        | 380 Hz (22,800 rpm) |

# Overspeed test of P2



## P2 after overspeed test



# Conclusions

- Chopper disk series NEAT II is “non-standard”
  - Large diameter
  - Deep windows
  - Unsymmetrical geometry
  - High rotational speed
- Many small problems added up and delayed the project
- New hub system is developed with high bending stiffness and low radial stiffness

# Thank you for your attention



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# Backup slides